

Review on Thermal Behaviour of Noncircular Orthogonal Journal Bearing

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Abstract – The insurgent changes appears in field of non-circular journal bearing from very sluggish speed to revved up turbo machinery. The growing demands from industries for revved up speed application and ability of circular bearing to support heavy loads have necessitated exercising the performance of bearing in detail under more prudent conditions. However, the revved up speed operations for bearing results in high temperature. Many researchers reported that increase in lubricant temperature is high in circular bearing. So it leads the researcher to consider hydrodynamic journal bearing having noncircular shape. The non-circular bearing shape modifies the journal stability under proper working conditions. This paper aims various studies gleam on orthogonally offset bearing.

Introduction

Noncircular journal bearing works at more than one active oil film. This attribute eminent stiffness and damping peculiarity of these bearing as compare to the circular journal bearing which operates at single active oil film. The unceasing progress of computer automation leads to designing of profile of non-circular bearing. In the midst of non-circular bearing, elliptical, three-lobe and offset half contours are frequent the most. Viscous drag increases the temperature in bearing due to which the viscosity of lubricant declined resulting in the falling of load capacity. The temperatures are determined through thermodynamic relations which also find the solution of Energy, Reynolds, Heat Conduction Equation. The film thickness is aperture among interior surface and journal through which lubricant flows. It is an essential parameter for Reynolds Equation, lubricant film temperature and thermal pressure. Non circularity on bearing surface is acquired by moving the two half-circle arc of the hover orthogonally in upward and descending of the center which makes even and vertical freedom, causes two distinct projections in bearing. Reynolds Equation is administering condition in oil execution investigation. The arrangement of the Reynold condition is picked up by Finite Difference Method utilizing focal distinction procedure. The temperature of the ointment is expanded because of the liquid erosion era in bearing. The deviation of temperature crosswise over film thickness in Energy Equation approximated by Parabolic Temperature Profile Approximation.

Literature Review

Pinkus and Lynn (1956) [1] first examine the steady state performance of the two lobe bearing, originate the basic peculiarity of film stiffness for bearings, too analyze the dynamic attributes for diverse oil films. Analysis of orthogonal shifted bearing is done by Willock (1961) [2] and studied stiffness is obtained without decreasing in film thickness and oil flow is increased and the temperature is less and performance is attained in particular order of rotation. In orthogonally displaced bearing one sententious stricture is received that upgraded performance is attained only in one order of revolution. Also investigated the circumstances of offsetting the centers of two lobe bearing in orthogonal tendency of mid radius of lobe. M. Malik et al (1982) [3] studied the systematic and broad data for offset-half journal bearing. Investigated the offset-half bearing for static and dynamic achievement attribute. Data reported in this paper for both laminar and turbulent system, offset factor 0.5, four aspect ratios (1/2, 1/3, 2/3 and 1). R.W. Wilson et al (1973) [4] investigated that various non observance that occurs in plain bearing which starts the researcher for farther survey on profile bearing. Reviewed the shortcomings regarding properties regarding the fitting, working environment, bearing material and also review decay of bearing surface due to lubricant, analyzed the metallurgical entities. D. F. Li et al (1980) [5] studied the stability characteristics of non-circular (four lobes) bearing and found that the offset bearing produces less vibration. Also erect that elliptical bearing has more stability for both heavy and medium load operations used, Fourier transformation analysis is value to achieve frequency content of nonlinear trajectory, results are better for even highly unstable bearings. High eccentricities desired positive bearing force reckoning fidelity, more elevated dynamic description below and above stability threshold of orthogonal bearing. Ajeet Singh et al (1982) [6] investigated that more the eccentricity ratio more shaft stability of elliptical bearing and also presented the data on several values of ellipticity, eccentricity and L/D ratios so that experiment achieves at balance zone, besides receives elliptical bearing are suitable for flexible and stiff rotors, shaft flexibility is unity or more in case the stability vicinity of elliptical bearing. N. P. Mehta et al (1987) [7] By offsetting horizontally centers of the two halves of elliptical pressure dam bearing investigated the stability of pressure

dam bearing is improved by deep grooves on upper and lower half surfaces, out of all non-circular bearing orthogonal bearing is more stable for low and high speed experiments, plots drag betwixt Sommerfeld number and dimensionless threshold spin propping rigid rotor, orthogonally offset bearing has exceedingly vast region of infinite stability. S. Basri and D.T. Gethin (1990) [8] analyzed thermal behavior of orthogonally offset bearing, have larger hydrodynamic requirement, minimum film temperature, and high stability but the power losses are more as compare to three and four lobe profiles. Thermal issue in profile bore bearing are inferior utmost than those clash in cylindrical bearing, load bear dexterity entanglement on thermal analysis, reliance on various attributes like spin of journal, eccentricity ratio and global working distinctive are introduced for four profile bore bearing, indicated the consequences of oil blending at grooves and temperature expedition in each lobe. In offset lobe bearing diminished circumferential alteration of oil film temperature occurs due to the bulky mass of cool lubricant in this type of bearing. Elevated power damage in support with four lobe bearing, eminent film temperature at cylindrical bearing, four lobe bearing has utmost lobe eccentricity ratio be accordant to least part film thickness. Ajeet Singh and B.K. Gupta (1982) [9] presented the research on orthogonally offset bearing and found that orthogonally displaced bearing is more stable than three-lobe and tilted three lobes bearing, when the suitable combination of horizontal and vertical clearances. Increasing L/D ratios bulge to unstable zone at high loading. Attitude angle and static load capacity computed for equilibrium is the summation of horizontal component of forces for both lobes is cipher, invariability limit of rigid and flexible rotors depends on ellipticity of noncircular bearing. A. Hussain et al (1996) [10] studied the thermal behavior of orthogonally displaced bearing, two lobe bearing, elliptical bearing using thermo-hydrodynamic analysis, found that orthogonal bearing has less temperature generation as compare to elliptical bearing but lower load capacity. Load capacity changes due to change in ellipticity ratios, observed that two lobe bearing has highest load carrying capacity, less for elliptical and more inferior for orthogonal bearing, temperature of the lubricant also depends on groove location, also observed that modeling of cavitation is a consequential feature in thermo hydrodynamic analysis as it affects the final temperature contour. L. Hyun Cheon Ha et al (1999) [11] investigated the frequency effects on the stiffness and damping on five pad tilting pad bearing, essential factor studied is excitation frequency, and controlled by orthogonally mounting hydraulic exciters, for acquirement of lineal behavior for the period of test, extent of excitation force is restrained, allotment of excitation frequency is relevant to movement betwixt bearing and shaft speed and also on acceleration of bearing casing, increase in damping coefficient in case of low speed and high load, stiffness coefficient leanly decline

as excitation frequency leanly augmented. Daen Wang et al (2004) [12] assess the oil execution for the non-roundabout bearing utilizing versatile hydro dynamic investigation, most extreme hydrodynamic weight is less in twisted bearing and high in the inflexible bearing, least oil film thickness and most prominent hydrodynamic weight in distorted interfacing pole bearing than unbending associating pole bearing, weight dissemination in oil covering is controlled by limited contrast method instead of logical permission, hydrodynamic weight is increasingly and film thickness is less for disfigured bearing than inflexible bearing. J. Sharna Basvaraja et al (2007) [13] two lobe hole entering bearing is having more stability than circular hole entering bearing, two lobe needs more pumping of lubricant as compare to circular, finite element and Galerkin's method used to solve the governing equations, large values of damping and stiffness coefficient is gained only when the displacement factor is more than one, displacement factor more than one procure augmented utility of stability threshold speed. Satish Jain et al (2010) [14] Two lobe multi recess hybrid journal bearing working varies as offset factor changes, journal and bearing arrangement counterpoised by capillary and orifice restrictors for universal bearing geometry profile, finite element method is appliance for computation of governing equations, performance depends upon the displacement factor of two lobe multi recess hybrid journal bearing, oil pumping is less required as compare circular bearing. Amit Chauhan et al (2010) [15] Examine the non-circular bearing (elliptical) using the different types of oil using thermo hydrodynamic analysis, found that rise pressure and lubricant temperature are the function of viscosity, temperature and pressure increases in the system with the increase of shaft speed and eccentricity ratio. Amit Singla et al. (2016) [16] analyzed the static behavior on non-circular bearing (elliptical and orthogonal profile), pressure rise in the orthogonal bearing is less as compare to an elliptical bearing, temperature generation in the orthogonal system is less than that to the elliptical bearing. Amit Chauhan et al (2011) [17] studied that the offset half bearing has less temperature formation as compared to the elliptical journal bearing, power loss is less and good load carrying as compare to an elliptical bearing. Cheng Chi Wang (2016) [18] study the three multi lobe air bearing using Finite Difference Method, attain the computational accuracy by using hybrid Finite Difference Method and Differential Transformation Method and get good efficiency and good accuracy. Amit Singla et al (2016) [19] experimentally study the performances of elliptical and orthogonal bearing, maximum temperature increment in orthogonal bearing is less as compare to the elliptical bearing, with the increase of radial load maximum flow rate, temperature and pressure in orthogonal bearing. Saroj Bala et al (2014) [20] studied the vertical offset journal bearing profile, two lobes on vertical offset bearing leads better thermal and pressure results, found that with gain in

shaft speed rise in oil coat temperature and deduce that offset bearing examined as interchange for circular bearing.

Conclusion

Thermo-hydrodynamic analysis for non-circular bearing has been tested either by simulation or experimentally using different approaches. All researches regarding the performance, stability, thermal pressure, lubricant temperature are feasible in case of the non-circular bearing when applying the proper boundary conditions and parameters to the bearing system. Increase in the ellipticity ratio for noncircular bearing there is decrease in the thermal pressure and oil temperature.

References

- [1] Pinkus, O. and Lynn, M.A. (1956), "Analysis of elliptical bearings", *Transaction of ASME*, Vol. 55 No. 22, pp. 965-973.
- [2] Willock, D.F. (1961), "Orthogonally displaced bearings-I" *ASLE Transaction*, Vol. 4 pp. 117-123, available at: www.maklubes.com/DisplaySelector/IndustrialProductDetails.aspx (accessed 2 June 2015).
- [3] M. Malik, Mahesh Chandra & R. Sinhasan (1982), "Design Data for Offset-Halves Journal Bearing in Laminar and Turbulent Regimes" *ASLE Transaction*, 25:1, 133-140, DOI: 10.1080/05698198208983075.
- [4] RW Wilson and EB Shone (1973), "The Diagnosis of Plain Bearing Failures" *Tribology Series* Vol.8, pp 80-131.
- [5] D. F. Li, K. C. Choy and P.E. Allaire,(1980), "Stability and Transient Characteristics of Four Multilobe Journal Bearing Configurations" *J. of Lubrication Tech* 102(3), 291-298 DOI: 10.1115/1.3251514.
- [6] Ajeet Singh and B.K. Gupta, (1982), Stability Limits of Elliptical Journal Bearing Supporting Flexible Rotors" *Wears*77 159-170.
- [7] Mehta, N.P. (1993),"Static and dynamic characteristics of orthogonally-displaced pressure dam bearings" *Tribology Transactions*, Vol. 36 No. 2, pp. 201-206.
- [8] S. Basri and D.T. Gethin. (1990), "A comparative study of the thermal behavior of profile bore bearings" *Tribology International* Volume 23, Issue 4, Pages 265-276.
- [9] Ajeet Singh and B.K. Gupta (1982), "Stability Limits of Elliptical Journal Bearings Supporting Flexible Rotors" *Wear* Volume 77, Issue 2 Pages 159-170.
- [10] A. Hussain, K. Mistry, S. Biswas, (1996), "Thermal Analysis of Noncircular Bearing" *Transaction ASME*, Vol. 118 No. 1 pp 246-254.
- [11] Hyun Cheon Ha, Seong Heon Yang, (1999), "Excitation Frequency Effects on the Stiffness and Damping Coefficients of a Five-Pad Tilting Pad Journal Bearing" *ASME J. Tribol* 121(3) 517-522.
- [12] Daen Wang and Theo G Keith, Qingmin Yang and Kumar Vaidyanathan, (2004) "Lubrication Analysis of a Connecting-Rod Bearing in a High-Speed Engine. Part II: Lubrication Performance Evaluation for Non-Circular Bearings" *Tribology Transaction* pp 290-298.
- [13] J. Sharana Basavaraja, Satish C. Sharma and S.C. Jain, (2007)"Performance of an Orifice Compensated Two-Lobe Hole-Entry Hybrid Journal Bearing" *Advances Tribology*, Article ID 871952, 10 pages.
- [14] Satish Jain, Satish Sharma, J. Sharana Basavaraja and Prashant Kushare, (2010) "Study of two-lobe four recessed hybrid journal bearing" *Industrial Lubrication and Tribology* ISSN: 0036-8792.
- [15] Amit Chauhan, Rakesh Sehgal, Rajesh Kumar Sharma, (2010) "Thermodynamic analysis of elliptical journal bearing with different grade oils", *Tribology International* Vol. 43, Issue 11 pp. 1970-1977.
- [16] Amit Singla and Amit Chauhan (2016) "Simulation studies on static thermal behavior of true elliptical and orthogonally displaced non-circular journal bearing" *Industrial Lubrication and Tribology* Vol. 68 Iss 3 pp.
- [17] Amit Chauhan, Rakesh Sehgal, Rajesh Kumar Sharma (2011), "Investigations on the thermal effects in non-circular journal bearings" *Tribology International* Vol.44 Issue 12 pp 1765-1773
- [18] Cheng-Chi Wang (2016), "Non-periodic and chaotic response of three-Multilobe air bearing system" *Applied Mathematical Modeling*.
- [19] Amit Singla and Amit Chauhan, (2016), "Experimental study for performance evaluation of steadily loaded true elliptical and orthogonally displaced non circular journal bearing profiles" *Industrial Lubrication and Tribology* ISSN: 0036-8792 Vol. 68 Issue 6.
- [20] Saroj Bala and Amit Chauhan (2014), "Simulation Studies on Vertical Offset Non Circular Journal Bearing Profile" *Tribology Online* ISSN: 1881-2198 DOI 10.2474/trol.9.31